

Course of "Automatic Control Systems" 2024/25

## Laplace transform, transfer function, Laplace domain analysis of LTIs

Prof. Francesco Montefusco

Department of Economics, Law, Cybersecurity, and Sports Sciences

Università degli studi di Napoli Parthenope

francesco.montefusco@uniparthenope.it

#### Team code: tz3jpwb



- ▲ Laplace transform definition
- ▲ Laplace transform main properties
- ▲ Selected Laplace transforms
- ▲ Transfer function definition
- ▲ LTI systems analysis in Laplace domain



The Laplace transform of a function 
$$f(t)$$
 is defined as  
or  $\mathcal{L}(f(t))$   
 $f(t) \to F(s) = L(f(t)) = \int_{0}^{+\infty} f(t)e^{-st}dt$ 

where  $t \in R$  is a real variable, while  $s = \alpha + j\omega \in C$  is a complex variable.

▲ Vice versa, given a function F(s) in the Laplace domain, the original function in the time domain can be obtained using the Laplace anti-transformation

$$F(s) \to f(t) = \lim_{\omega \to \infty} \frac{1}{2\pi j} \int_{\sigma-j\omega}^{\sigma+j\omega} F(s) e^{st} ds$$

A The Laplace transform is a bilateral only if the function f(t) is null for t < 0



#### *▲ Linearity*

$$L(af(t)+bg(t)) = aF(s)+bG(s)$$

#### ▲ Translation in the Laplace domain

$$L(e^{\alpha t}f(t)) = F(s-\alpha)$$

▲ Translation in the time domain

$$L(f(t-T)) = F(s)e^{-sT}$$



## Laplace transform main properties (2/2)

*▲ Time domain derivation* 

$$L\left(\frac{df(t)}{dt}\right) = sF(s) - f(0)$$

*▲ Time domain integration* 

$$L\left(\int_{0}^{t} f(\tau) d\tau\right) = \frac{1}{s}F(s)$$

▲ *Time domain convolution* 

$$L(f(t) * g(t)) = F(s)G(s)$$



#### ▲ Initial value theorem

$$f(0) = \lim_{s \to \infty} sF(s)$$

▲ Final value theorem

$$\lim_{t\to\infty} f(t) = \lim_{s\to 0} sF(s)$$

▲ Initial value theorem of the derivate of the function

$$\frac{df(t)}{dt}\Big|_{t=0} = \lim_{s \to \infty} s^2 F(s) - sf(0)$$



▲ In the system theory, we will mainly use the Laplace transform for the evaluation of the forced response of LTI systems to selected sets of input :

\* Polynomial inputs  $u(t) = t^n \mathbf{1}(t)$ 

\* Sinusoidal inputs  $u(t) = sin(\omega t) \mathbf{1}(t)$  $u(t) = cos(\omega t)\mathbf{1}(t)$ 



## Selected Laplace transforms



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# Selected Laplace transforms: polynomial signals

▲ In order to evaluate the Laplace transform of polynomial signals, let us firstly consider the Laplace transform of the impulse

- \* Impulse  $\delta(t) \longrightarrow L(\delta(t)) = 1$  (from the Laplace transform definition)
- ▲ Then, using the *time domain integration property*, we have

$$\Rightarrow Step \quad 1(t) \quad \longrightarrow \quad L(1(t)) = \frac{1}{s}$$

$$\Rightarrow Ramp \quad t \cdot l(t) \quad \longrightarrow \quad L(t \cdot l(t)) = \frac{1}{s^2}$$

\* Polinomial function 
$$t^n \cdot l(t) \longrightarrow L(t^n \cdot l(t)) = \frac{n!}{s^{n+1}}$$



# Selected Laplace transforms: sinusoidal signals

The Laplace transform of sinusoidal functions  $\Rightarrow Sine \quad sin(\omega t)\mathbf{1}(t) \longrightarrow L(sin(\omega t)\cdot\mathbf{1}(t)) = \frac{\omega}{s^2 + \omega^2}$ 

$$\Rightarrow Cosine \quad cos(\omega t)\mathbf{1}(t) \quad \longrightarrow \quad L(cos(\omega t)\cdot\mathbf{1}(t)) = \frac{s}{s^2 + \omega^2}$$

▲ Finally, in the control theory the following transformations are of interest for the definition of the Laplace domain of the evolution modes of LTI systems

$$L(e^{\alpha t} 1(t)) = \frac{1}{s - \alpha}$$
$$L(e^{\alpha t} \cos(\omega t) \cdot 1(t)) = \frac{s - \alpha}{(s - \alpha)^2 + \omega^2}$$
$$L(e^{\alpha t} sen(\omega t) \cdot 1(t)) = \frac{\omega}{(s - \alpha)^2 + \omega^2}$$



## Example: Laplace transform of a window signal





▲ The Laplace transform of a window signal can be evaluated from the Laplace transforms of two steps.

$$L(u(t)) = L(1(t) - 1(t - T))$$
  
=  $L(1(t)) - L(1(t - T))$   
=  $\frac{1}{s} - \frac{e^{-sT}}{s} = \frac{1 - e^{-sT}}{s}$ 



# Solution of first order linear differential equation

Let us consider a first order differential equation, linear with constant coefficients,

$$\dot{y}(t) + a_0 y(t) = b_0 u(t), \quad y(t_0) = y_0$$

By applying Laplace trasform, assuming a step input signal,  $u(t)=U_01(t)$ , with amplitude  $U_0$  $L(\dot{y}(t) + a_0y(t)) = L(b_0U_01(t))$ 





# Solution of first order linear differential equation

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$$Y_{free}(s) = \frac{y_0}{s + a_0} \stackrel{\mathcal{L}^{-1}}{\longrightarrow} \qquad y_{free}(t) = e^{-a_0 t} y_0 \mathbf{1}(t)$$
$$Y_{forced}(s) = \frac{b_0 U_0}{s(s + a_0)} = \frac{A}{s} + \frac{B}{s + a_0}$$

Compute *A* and *B* by equating:

$$Y_{forced}(s) = \frac{A(s + a_0) + Bs}{s(s + a_0)}$$
  
=  $\frac{(A + B)s + Aa_0}{s(s + a_0)}$   
 $A = \frac{b_0 U_0}{a_0}$   
 $A = \frac{b_0 U_0}{a_0}$   
 $B = -\frac{b_0 U_0}{a_0}$ 

Or by residual method:

$$A = (s - 0)Y_{forced}(s)|_{s=0}$$
$$= \frac{b_0 U_0}{s + a_0}|_{s=0} = \frac{b_0 U_0}{a_0}.$$

$$B = (s - (-a_0))Y_f(s)|_{s=-a_0}$$
$$= \frac{b_0 U_0}{s}|_{s=-a_0} = -\frac{b_0 U_0}{a_0}.$$

 $\left|\right|$ 



# Solution of first order linear differential equation

$$Y_{forced}(s) = \frac{b_0 U_0}{s(s+a_0)} = \frac{A}{s} + \frac{B}{s+a_0} = \frac{\frac{b_0 U_0}{a_0}}{s} + \frac{-\frac{b U_0}{a_0}}{s+a_0}$$
$$\mathcal{L}^{-1}$$
$$y_{forced}(t) = \frac{b_0 U_0}{a_0} 1(t) - \frac{b_0 U_0}{a_0} e^{-a_0 t} 1(t) = \frac{b_0 U_0}{a_0} (1 - e^{-a_0 t}) 1(t)$$

Then,

$$y(t) = y_{free}(t) + y_{forced}(t) = \left(e^{-a_0 t}y_0 + \frac{b_0 U_0}{a_0}(1 - e^{-a_0 t})\right) \mathbf{1}(t)$$



## Laplace Transform and Transfer function

- ▲ The analysis of LTI system is simplified by using Laplace transform.
- A By exploiting the important property of the Laplace transform of the derivative of a signal f(t) (with zero initial conditions, i.e. f(0) = 0)

$$\mathcal{L}\left(\dot{f}(t)\right) = sF(s),$$

A Given the differential equation of a linear system, it is possible to find the transfer function, G(s), of that system, defined by

$$G(s) = \frac{Y(s)}{U(s)}$$

▲ Then for a LTI system of first order described by

$$\mathcal{L} \bigvee \dot{y}(t) + a_0 y(t) = b_0 u(t), y(0) = y_0 = 0$$
  

$$sY(s) + a_0 Y(s) = b_0 U(s)$$
  

$$Y(s)(s + a_0) = b_0 U(s)$$
  

$$G(s) = \frac{Y(s)}{U(s)} = \frac{b_0}{s + a_0}$$



## Laplace Transform and Transfer function

#### ▲ Then, for a LTI system of second order described by

$$\ddot{y}(t) + a_1 \dot{y}(t) + a_0 y(t) = b_0 u(t), y(0) = 0, \dot{y}(0) = 0$$

$$s^{2}Y(s) + a_{1}sY(s) + a_{0}Y(s) = b_{0}U(s)$$
$$Y(s)(s^{2} + a_{1}s + a_{0}) = b_{0}U(s)$$
$$G(s) = \frac{Y(s)}{U(s)} = \frac{b_{0}}{s^{2} + a_{1}s + a_{0}}$$

A Therefore, given the transfer function G(s) and the input u(t) with transfer function U(s), the output is the product

$$Y(s) = G(s)U(s)$$

▲ Using Laplace transforms, the output Y(s) can be expanded into its elementary terms (i.e., the sum of well-known transforms of selected/common signals/functions) by partial fraction expansion: therefore, it is possible to rewrite Y(s) as a sum of terms for which is known the anti-transformation; the total time function y(t) is given by the sum of these anti-transformation terms.



## Laplace Transform and Transfer function

Using Laplace transforms, the output Y(s) can be expanded into its elementary terms (i.e., the sum of well-known transforms of selected/common signals/functions) by partial fraction expansion: therefore, it is possible to rewrite Y(s) as a sum of terms for which is known the anti-transformation; the total time function y(t) is given by the sum of these anti-transformation terms.

$$Y(s) = Y_1(s) + Y_2(s) + \cdots$$

$$\mathcal{L}^{-1} \qquad \mathcal{L}^{-1} \qquad$$

 $Y_i$  as,

$$Y_i(s) = \frac{A \mathcal{L}^{-1}}{s} y_i(t) = A \cdot 1(t); \qquad Y_i(s) = \frac{A}{s-\alpha} \xrightarrow{\mathcal{L}^{-1}} y_i(t) = A \cdot e^{\alpha t} 1(t)$$

$$Y_i(s) = \frac{\omega}{(s-\alpha)^2 + \omega^2} \xrightarrow{\mathcal{L}^{-1}} y_i(t) = e^{\alpha t} \sin(\omega t) \, 1(t)$$
$$Y_i(s) = \frac{s-\alpha}{(s-\alpha)^2 + \omega^2} \xrightarrow{\mathcal{L}^{-1}} y_i(t) = e^{\alpha t} \cos(\omega t) \, 1(t)$$



▲ Let us consider a Linear Time Invariant (LTI) system in the state space form

$$\dot{x}(t) = Ax(t) + Bu(t), \quad x(t_0) = x_0$$
 (1.a)

$$y(t) = Cx(t) + Du(t)$$
(1.b)

A The Evaluation of an LTI system response in a transformed domain is convenient only if





▲ Let us indicate with X(s), U(s) and Y(s) the Laplace transforms of the signals x(t), u(t) and y(t).

Transforming both the sides of the equation (1a),

$$L(\dot{x}(t)) = L(Ax(t) + Bu(t))$$

using the time domain derivation property of the Laplace transform, a linear system in the Laplace domain can be written has

$$sX(s) - x_0 = AX(s) + BU(s)$$

we have

$$(sI - A)X(s) = x_0 + BU(s)$$

and

$$X(s) = (sI - A)^{-1}x_0 + (sI - A)^{-1}BU(s)$$

The matrix function  $\Phi(s) = (sI - A)^{-1}$  is called *Transition matrix*, then

$$X(s) = \Phi(s)x_0 + \Phi(s)BU(s)$$



▲ Given a quadratic and invertible matrix

$$A = \begin{pmatrix} x_{1,1} & \dots & x_{1,j} \\ \vdots & \ddots & \vdots \\ x_{i,1} & \dots & x_{i,j} \end{pmatrix}$$

its inverse is defined as

$$A^{-1} = \frac{1}{\det(A)} \begin{pmatrix} \operatorname{cof}(A, x_{1,1}) & \dots & \operatorname{cof}(A, x_{1,j}) \\ \vdots & \ddots & \vdots \\ \operatorname{cof}(A, x_{i,1}) & \dots & \operatorname{cof}(A, x_{i,j}) \end{pmatrix}^T$$

where the cofactor is

$$\operatorname{cof}(A, i, j) = (-1)^{i+j} \operatorname{det}(\operatorname{minor}(A, i, j))$$

and the minor (i, j) is the determinant of the matrix obtained excluding the row i and the column j.



### Inverse of a 2×2 matrix

 $\checkmark$  Given a matrix

$$A = \begin{pmatrix} a & b \\ c & d \end{pmatrix}$$

its inverse is

$$A^{-1} = \frac{1}{ad - bc} \begin{pmatrix} d & -b \\ -c & a \end{pmatrix}$$



#### Inverse of a 3×3 matrix

#### $\blacktriangle$ Given a matrix

$$A = \begin{pmatrix} A_{11} & A_{12} & A_{13} \\ A_{21} & A_{22} & A_{23} \\ A_{31} & A_{32} & A_{33} \end{pmatrix}$$

its inverse is

$$\frac{1}{\det(A)} \begin{pmatrix} + \begin{vmatrix} A_{22} & A_{23} \\ A_{32} & A_{33} \end{vmatrix} & - \begin{vmatrix} A_{12} & A_{13} \\ A_{32} & A_{33} \end{vmatrix} & + \begin{vmatrix} A_{12} & A_{13} \\ A_{22} & A_{23} \end{vmatrix}} \\ - \begin{vmatrix} A_{21} & A_{23} \\ A_{31} & A_{33} \end{vmatrix} & + \begin{vmatrix} A_{11} & A_{13} \\ A_{31} & A_{33} \end{vmatrix} & - \begin{vmatrix} A_{11} & A_{13} \\ A_{21} & A_{23} \end{vmatrix} \\ + \begin{vmatrix} A_{21} & A_{22} \\ A_{31} & A_{32} \end{vmatrix} & - \begin{vmatrix} A_{11} & A_{12} \\ A_{31} & A_{32} \end{vmatrix} & + \begin{vmatrix} A_{11} & A_{12} \\ A_{31} & A_{32} \end{vmatrix}$$



#### Transition matrix

- For the *Transition matrix*  $\Phi(s) = (sI A)^{-1}$ 
  - $\circ$  Each element is a rational function in *s* variable:
    - ➤ denominator of degree *n* given by  $det(sI A) = p_A(s)$ , whose roots are the eigenvalues of A.
    - Inumerator of element (i,j) corresponds to the algebraic complement of element (j,i) which by construction is a degree at most *n*-1



- ▲ Transforming both the sides of the equation (1b), we have  $L(y(t)) = L(Cx(t) + Du(t)) \Leftrightarrow Y(s) = CX(s) + DU(s)$
- A and by substituting the previous equation,  $X(s) = \Phi(s)x_0 + \Phi(s)BU(s)$   $Y(s) = C\Phi(s)x_0 + C\Phi(s)BU(s) + DU(s)$  $Y(s) = C\Phi(s)x_0 + (C\Phi(s)B + D)U(s)$
- ▲ The matrix function  $G(s) = C\Phi(s)B + D = C(sI A)^{-1}B$  is called *transfer function*, therefore

$$Y(s) = C\Phi(s)x_0 + G(s)U(s)$$

- ▲ For *Single Input Single Output (SISO)* systems the transfer function G(s) is a scalar function;
- ▲ For *Multiple Input Multiple Output (MIMO)* systems the transfer function G(s) is a matrix whose element  $G(s)_{ij}$  will connect the output *i* with the input *j*.



#### Transfer function

• For the *Transfer function*  $G(s) = C\Phi(s)B + D = C(sI - A)^{-1}B + D$ 

$$\boldsymbol{\Phi}_{ij}(s) = \frac{N(s)}{D(s)}$$

N(s) of degree at most n-1D(s) of degree n,

G(s) is a rational function in s variable:

$$G(s) = \frac{a_m \, s^m + a_{m-1} \, s^{m-1} + \dots + a_1 \, s + a_0}{b_n \, s^n + a_{n-1} \, s^{n-1} + \dots + b_1 \, s + b_0}$$

- Since the multiplication on the left of  $\Phi(s)$  by *C* and the one on the right by *B* correspond to a linear combination of  $\Phi(s)$  elements, all with the same denominator, i.e. det(sI A), then all the elements of  $C\Phi(s)B$  are rational functions in *s* with a denominator polynomial of degree *n* and a numerator of degree  $m \le n 1$ :
- > If D=0, m < n, the system is said strictly proper.

$$\blacktriangleright \text{ If } D \neq \mathbf{0}, m = n, \text{ the system is said proper.}$$



$$\dot{x} = \begin{pmatrix} -2 & -1.5 \\ 2 & 0 \end{pmatrix} x + \begin{pmatrix} 2 \\ 0 \end{pmatrix} u \qquad \dot{x} = \begin{pmatrix} -4 & -2.5 \\ 2 & 0 \end{pmatrix} x + \begin{pmatrix} 2 \\ 0 \end{pmatrix} u y = (-1.5 & -1.25)x + u$$
$$\mathbf{y} = (-1.5 & -1.25)x + u$$



#### Transfer function

▲ Given a *transfer function* 

$$G(s) = \frac{N(s)}{D(s)} = \frac{a_m s^m + a_{m-1} s^{m-1} + \dots + a_1 s + a_0}{b_n s^n + a_{n-1} s^{n-1} + \dots + b_1 s + b_0}$$

- A The roots of the N(s) are said *zeros*.
- A The roots of the D(s) are said *poles*.
- A The polynomial D(s) is defined as D(s) = det(sI A), hence
  - D(s) coincides with the characteristic polynomial of the system
  - *the poles coincide with the eigenvalues of the system* except for possible pole-zero cancellation



## Then, for a LTI system, by Laplace transform the state equation: $L(\dot{x}(t)) = L(Ax(t) + Bu(t)), \quad x(t_0) = x_0 \implies$ $sX(s) - x_0 = AX(s) + BU(s) \implies (sI - A)X(s) = x_0 + BU(s)$ $X(s) = (sI - A)^{-1}x_0 + (sI - A)^{-1}BU(s)$ $X(s) = \Phi(s)x_0 + \Phi(s)BU(s)$ Transition matrix

By Laplace transform the output equation: L(y(t)) = L(Cx(t) + Du(t))

$$Y(s) = CX(s) + DU(s)$$
  
=  $C\Phi(s)x_0 + C\Phi(s)BU(s) + DU(s)$   
=  $C\Phi(s)x_0 + (C\Phi(s)B + D)U(s)$ 

**G(s): transfer function** 

 $Y_{free}(s)$ 

 $Y(s) = C\Phi(s)x_0$ 

+G(s)U(s)



▲ For SISO systems the free evolution in the Laplace domain is given by the ratio of polynomial functions

$$Y_{free}(s) = C\Phi(s)x_0$$

This is also true for the forced evolution in case we restrict our attention to the case of polynomial and sinusoidal inputs

$$Y_{forced}(s) = G(s)U(s)$$

A It is convenient to antitransform Y(s) by reducing the ratio of high degree polynomial functions to the sum of selected signals transform such as

$$L(e^{\alpha t}\cos(\omega t)\cdot 1(t)) = \frac{s-\alpha}{(s-\alpha)^2 + \omega^2} \qquad L(e^{\alpha t}sen(\omega t)\cdot 1(t)) = \frac{\omega}{(s-\alpha)^2 + \omega^2}$$
$$L(e^{\alpha t}1(t)) = \frac{1}{s-\alpha}$$



### Laplace antitransform

Different methods can be used to reduce the ratio of high degree polynomial functions to the sum polynomial functions of degree one or two, such as *the residual method* (see the book for details).

Residual method for real and distinct eigenvalues

(see the book for the other cases)

$$F(s) = \frac{N(s)}{D(s)} = \frac{N(s)}{\prod_{i=1}^{n} (s-p_i)} \qquad p_i \neq p_j \text{ for } i \neq j$$

▲ In case of real and distinct eigenvalues, F(s) can be also written as

$$F(s) = \sum_{i=1}^{n} \frac{A_i}{s - p_i}$$

where 
$$A_k = \lim_{s \to p_k} (s - p_k)F(s)$$
. Hence

$$f(t) = \sum_{i=1}^{n} A_i e^{p_i t}$$



## LTI system, first order, strictly proper (d=0)

Consider a first order differential equation, linear with constant coefficients,

$$\dot{y}(t) + a_0 y(t) = b_0 u(t), \quad y(t_0) = y_0$$

that can be described by LTI system as

$$\dot{x}(t) = ax(t) + bu(t), \quad x(t_0) = x_0$$

$$y(t) = x(t)$$

Note that  $b = b_0$ ,  $a = -a_0$ ,  $x_0 = y_0$ 

By Laplace transform

$$Y(s) = X(s) = \Phi(s)x_0 + G(s)U(s),$$

Where



Then,



## LTI system, first order, strictly proper: free and force responses

$$Y_{free}(s) = \frac{y_0}{s-a} \xrightarrow{\mathcal{L}^{-1}} y_{free}(t) = e^{at}y_0 1(t) = e^{-\frac{1}{a}} \frac{t}{\tau} y_0 1(t)$$

$$u(t) = U_0 1(t)$$
  

$$Y_{forced}(s) = \frac{bU_0}{s(s-a)} = \frac{A}{s} + \frac{B}{s-a}$$
  
Compute A and B by imposing  
the equality :

$$Y_{forced}(s) = \frac{A(s-a) + Bs}{s(s-a)}$$
$$= \frac{(A+B)s - Aa}{s(s-a)}$$
$$A = \frac{bU_0}{-a}$$
$$A = \frac{bU_0}{-a}$$
$$B = \frac{bU_0}{a}$$

Or by residual method:

$$A = (s - 0)Y_{forced}(s)|_{s=0}$$
  
=  $\frac{bU_0}{s - a}|_{s=0} = \frac{bU_0}{-a}.$ 

$$B = (s - a)Y_f(s)|_{s=a}$$
$$= \frac{bU_0}{s}|_{s=a} = \frac{bU_0}{a}.$$

 $\left|\right|$ 



### LTI system, first order, strictly proper: free and force responses

$$Y_{forced}(s) = \frac{bU_0}{s(s-a)} = \frac{A}{s} + \frac{B}{s-a} = \frac{\frac{bU_0}{-a}}{s} + \frac{\frac{bU_0}{a}}{s-a}$$

By denoting with 
$$G_0 = \frac{b}{-a}$$
,  $\tau = -\frac{1}{a}$   $Y_{forced}(s) = \frac{G_0 U_0}{s} - \frac{G_0 U_0}{s - a}$ 

Then,

$$y(t) = y_{free}(t) + y_{forced}(t) = \left(e^{-\frac{t}{\tau}}y_0 + G_0U_0(1 - e^{-\frac{t}{\tau}})\right) \mathbf{1}(t)$$



# LTI system, first order, strictly proper: free response





# LTI system, first order, strictly proper: step response





Response to a step input for a first order, strictly proper system ( $G_0 = 1$ ,  $\tau = 1$ )





LTI system, first order, strictly proper: parameters for the qualitative step response

- ▲ Initial value y(0) = 0
- $\checkmark Final value \ \lim_{t\to\infty} y(t) = G_0 U_0$
- *▲* Settling time
  - $t_{s 5\%} = 3\tau$   $t_{s 1\%} = 4.6\tau$
- A Rise time  $t_r \cong 2.2\tau$



### Example: mass-spring-damper system



• Input output representation

 $u(t)=f(t) \qquad M\ddot{y}(t) + B\dot{y}(t) + Ky(t)=u(t)$ y(t)=s(t)

• State space representation  $x_1 = s e x_2 = v = ds/dt$ 

$$\dot{x} = \begin{pmatrix} \dot{x}_1 \\ \dot{x}_2 \end{pmatrix} = \begin{pmatrix} 0 & 1 \\ -\frac{K}{M} & -\frac{B}{M} \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \end{pmatrix} + \begin{pmatrix} 0 \\ 1/M \end{pmatrix} u,$$
$$y = (1 \quad 0) \begin{pmatrix} x_1 \\ x_2 \end{pmatrix}$$



# In general a second order system...as a mass-spring-damper system

• Input output representation

 $\ddot{y}(t) + a_1 \dot{y}(t) + a_0 y(t) = bu(t)$ 

• State space representation  $x_1 = y \in x_2 = \dot{y} = \frac{dy}{dt}$   $\begin{pmatrix} x_1 \\ x_2 \end{pmatrix} = \begin{pmatrix} y \\ \dot{y} \end{pmatrix}$ 

$$\dot{x} = \begin{pmatrix} \dot{x}_1 \\ \dot{x}_2 \end{pmatrix} = \begin{pmatrix} 0 & 1 \\ -a_0 & -a_1 \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \end{pmatrix} + \begin{pmatrix} 0 \\ b \end{pmatrix} u,$$
$$y = (1 \quad 0) \begin{pmatrix} x_1 \\ x_2 \end{pmatrix}$$

• Transition matrix

• Transfer function

$$\Phi(s) = \frac{\begin{pmatrix} s+a_1 & 1\\ -a_0 & s \end{pmatrix}}{s^2 + a_1 s + a_0}$$

$$G(s) = \frac{b}{s^2 + a_1 s + a_0}$$

$$Y(s) = C\Phi(s)x_0 + G(s)U(s)$$



In general a second order system... as a mass-spring-damper system

$$Y(s) = C\Phi(s)x_0 + G(s)U(s) \qquad x_0 = \begin{pmatrix} x_{10} \\ x_{20} \end{pmatrix}$$
$$Y(s)_{free} \qquad Y(s)_{forced}$$

$$Y(s)_{free} = \frac{(s+a_1)x_{10} + x_{20}}{s^2 + a_1 s + a_0}$$
$$U(s) = \frac{U_0}{s}, \quad Y(s)_{step} = \frac{b}{s^2 + a_1 s + a_0} \cdot \frac{U_0}{s}$$



### Characteristic equation ...

• The characteristic equation,  $s^2 + a_1s + a_0 = 0$ , determines the evolution modes

### **Three cases:**

- real and distinct poles
- real multiple poles
- complex conjugates poles



> Real and distinct poles, terms as  $\frac{1}{s-a}$ corresponding to a real pole/real eiger

corresponding to a real pole/real eigenvalue of the dynamic matrix

> Real multiple poles, a term as

$$\frac{1}{(s-a)^2}$$

1

corresponding to a real multiple pole/eigenvalue of the dynamic matrix

> Complex conjugates poles, terms as

$$\frac{\omega}{(s-\alpha)^2+\omega^2}$$
 or  $\frac{s-\alpha}{(s-\alpha)^2+\omega^2}$ 

corresponding to a pair of complex conjugate poles/eigenvalues  $a \pm j\omega$  of the dynamic matrix



CASE 1: real and distinct eigenvalues/poles

$$Y_{free}(s) = C\Phi(s)x_0 = \frac{s-10}{s^2+7s+10} = \frac{s-10}{(s+2)(s+5)}$$

▲ Appling the residual method we have

$$Y_{free}(s) = \frac{A_1}{(s+2)} + \frac{A_2}{(s+5)}$$

with

$$A_{1} = \lim_{s \to -2} (s+2)Y_{free}(s) = \lim_{s \to -2} \frac{s-10}{s+5} = -4$$
$$A_{2} = \lim_{s \to -5} (s+5)Y_{free}(s) = \lim_{s \to -5} \frac{s-10}{s+2} = 5$$

Hence

$$y_{free}(t) = (-4e^{-2t} + 5e^{-5t}) \cdot 1(t)$$



CASE 2: real multiple eigenvalues/poles

$$Y_{forced}(s) = G(s)U(s) = \frac{18}{s^2 + 6s + 9}U(s)$$
 with  $u(t) = 1(t)$ 

 $\checkmark$  This function can be written as the sum of three terms

$$Y_{forced}(s) = \frac{18}{s(s+3)^2} = \frac{A_1}{s} + \frac{A_2}{(s+3)} + \frac{A_3}{(s+3)^2}$$

A The residual method can be applied to evaluate  $A_1$  and  $A_3$ , while  $A_2$  can be evaluated by substitution

$$\begin{aligned} A_1 &= \lim_{s \to 0} sY_{forced}(s) = 2 & A_3 = \lim_{s \to -3} (s+3)^2 Y_{forced}(s) = -6 \\ \text{while } A_2 &= -2. \\ \text{Hence,} & y_{forced}(t) = (2 - 2e^{-3t} - 6te^{-3t}) \cdot 1(t) \end{aligned}$$



▲ Note, in the case of multiple poles (with multiplicity  $r_i$ ),

$$F(s) = \frac{N(s)}{(s-p_i)^{r_i}}$$

$$F(s) = \frac{N(s)}{(s-p_i)^{r_i}} = \sum_{l=1}^{r_i} \frac{K_{il}}{(s-p_i)^{r_i-l+1}}$$

$$K_{i\ell} = \frac{1}{(\ell-1)!} \frac{d^{\ell-1}}{ds^{\ell-1}} (s-p_i)^{r_i} F(s)|_{s=p_i}$$

$$f(t) = \sum_{l=1}^{r_i} \frac{K_{il}}{(r_i-l)!} t^{r_i-l} e^{p_i t}$$



### Laplace antitransform: example 2

....CASE 2: real multiple eigenvalues/poles

For

$$Y_{forced}(s) = \frac{18}{s(s+3)^2} = \frac{A_1}{s} + \frac{A_2}{(s+3)} + \frac{A_3}{(s+3)^2}$$

 $A_2$  can be computed through residual method by

$$A_{2} = \lim_{s \to 0} \frac{d}{ds} (s+3)^{2} Y_{forced}(s) = \frac{d}{ds} (s+3)^{2} Y_{forced}(s)|_{s=-3} = \frac{d}{ds} \frac{18}{s}|_{s=3} = -\frac{18}{s^{2}}|_{s=3} = -2$$



#### CASE 3: complex conjugate eigenvalues/poles

$$Y_{free}(s) = C\Phi(s)x_0 = \frac{s+3}{(s^2+4s+13)}$$

 $\checkmark$  This function can be written as the sum of two terms

$$Y_{free}\left(s\right) = \frac{s+3}{\left(s^2+4s+4+9\right)} = \frac{s+2-2+3}{\left((s+2)^2+3^2\right)} =$$

A Hence, 
$$Y_{free}(s) = \frac{s+2}{(s+2)^2+3^2} + \frac{1}{3}\frac{3\cdot 1}{(s+2)^2+3^2}$$

$$y_{free}(t) = \left(e^{-2t}\cos(3t) + \frac{1}{3}e^{-2t}\sin(3t)\right) \cdot 1(t)$$
$$y_{free}(t) = e^{-2t}\left(\cos(3t) + \frac{1}{3}\sin(3t)\right) \cdot 1(t)$$



- A linear system is said *stable* if no evolution mode is divergent (only convergent and constant evolution modes).
- ▲ It happens if all the eigenvalues of the matrix A (pole of G(s)) have a negative or null real part and the eigenvalues with null real part have multiplicity 1.
- ▲ In a stable system
  - the free evolution doesn't tend to infinity
  - *the free evolution doesn't converge to zero* if the constant evolution mode is excited



- ▲ A linear system is said *asymptotically stable* if all evolution modes are convergent.
- A It happens if all the eigenvalues of the matrix A (pole of G(s) have negative real part
- ▲ In an *asymptotically stable* system
  - the free evolution converges to zero





- ▲ A linear system is said *unstable* if there is a divergent evolution mode.
- ▲ It happens if an eigenvalues of the matrix A (pole of G(s) have a real part positive or an eigenvalue (pole) with null real part with multiplicity >1.
- ▲ In an *unstable* system
  - \* the free evolution tends to infinity



▲ Let us consider the LTI system

$$\dot{x} = \begin{bmatrix} 0 & 1 \\ -1 & -2 \end{bmatrix} x + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u$$
$$y = \begin{bmatrix} 1 & 0 \end{bmatrix} x$$

- A In order to evaluate the eigenvalues of the matrix A, we can calculate the roots of the characteristic polynomial
- ▲ In Matlab, it is possible to use the command eig(A)
- ▲ In this example we have  $p_1 = p_2 = -1$ .
- This system is *asymptotically stable* because it has all eigenvalues with negative real part



▲ Let us consider the LTI system

$$\dot{x} = \begin{bmatrix} 0 & 1 \\ -1 & 2 \end{bmatrix} x + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u$$
$$y = \begin{bmatrix} 1 & 0 \end{bmatrix} x$$

- ▲ In this example we have  $p_1 = p_2 = 1$ .
- ▲ The system is *unstable* because it has two eigenvalues with positive real part



▲ Let us consider the LTI system

$$\dot{x} = \begin{bmatrix} 0 & 1 \\ 0 & -1 \end{bmatrix} x + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u$$
$$y = \begin{bmatrix} 1 & 0 \end{bmatrix} x$$

▲ In this example we have  $p_1 = 0$  ,  $p_2 = -1$ .

- ▲ The system is *stable* because it has
  - ▲ a null eigenvalue
  - $\blacktriangle$  an eigenvalue with negative real part



▲ Let us consider the transfer function of an LTI system

$$G(s) = \frac{s+1}{s^2(s+5)}$$

▲ This system is *unstable* because it has two null poles.



- Routh-Hurwitz criterion is used to study the sign of the real part of a polynomial roots.
- ▲ It is particularly useful in case of high order polynomials or polynomials with uncertain parameters where the evaluation of the roots can be difficult.
- Routh-Hurwitz criterion is of interest to study the stability of LTI systems both in the state-space form and in the Laplace domain

$$W(s) = \frac{b_m s^m + b_{m-1} s^{m-1} + \ldots + b_0}{a_n s^n + a_{n-1} s^{n-1} + \ldots + a_0}$$

where the poles of W(s) coincide with the eigenvalues of the matrix A



▲ Let us consider a polynomial

$$D(s) = a_n s^n + a_{n-1} s^{n-1} + \ldots + a_0$$

and without loss of generality let us assume that



$$*a_0 \neq 0$$

▲ <u>Stodola criterion (Necessary condition):</u>

A necessary condition for the roots of the polynomial D(s) to have negative real parts is that

$$sign(a_0) = sign(a_1) = \cdots = sign(a_n).$$

This condition is also sufficient for polynomials of degree n = 1, n = 2.





▲ Let us consider the polynomial

$$D(s) = a_n s^n + a_{n-1} s^{n-1} + \ldots + a_0$$

▲ *The Routh table* is defined as follows





- A Routh table, n + 1 rows and the last row has 1 element different from zero.
- ▲ Let us define the Routh table of the function





#### Routh criterion

#### $\checkmark$ Let us consider the Routh table of the polynomial D(s)

$$n$$
 $a_n$ 
 $a_{n-2}$ 
 $a_{n-4}$ 
 ...

  $n-1$ 
 $a_{n-1}$ 
 $a_{n-3}$ 
 $a_{n-5}$ 
 ...

  $n-2$ 
 $b_{n-2}$ 
 $b_{n-4}$ 
 $b_{n-6}$ 
 ...

  $n-3$ 
 $c_{n-3}$ 
 $c_{n-5}$ 
 ...
 ...

 ...
 ...
 ...
 ...
 ...

- A The roots of the polynomial D(s) have all negative real parts iff the elements of the first column of the Routh table are all positive.
- A Each sign variation of the element of the first column of the Routh table correspond to a root of D(s) with a positive real part.



### Routh criterion: example

 $\checkmark$  Let us consider the polynomial

$$f(s) = s^4 + 2s^3 + 3s^2 + 5s + 10$$

A The Routh table of f(s) is





▲ Let us consider a transfer function W(s) of an LTI system where the poles of W(s) depends on un uncertain parameter p,

$$W(s) = \frac{s+1}{2s^3 + 5ps^2 + (3+p)s + 1}$$

▲ From the Routh table we have that





- ▲ In the design of the Routh table two singular cases can be found
  - a) The first term of a row is null
  - b) All the terms of a row are null
- ▲ In these cases, some mathematical manipulations of the Routh table can be adopted. However, it is not of interest for this course.