



Course of
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Analysis of LTI systems in the time domain

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LTI systems in the time domain

✦ *Linear time invariant (LTI) systems* in the form

$$\begin{aligned} \dot{x}(t) &= Ax(t) + Bu(t) \\ y(t) &= Cx(t) + Du(t), \end{aligned} \quad x(t_0) = x_0$$

with $A \in R^{n \times n}$, $B \in R^{n \times m}$, $C \in R^{p \times n}$, $D \in R^{p \times m}$, where $x(t)$ is the state vector, $u(t)$ is the input vector and $y(t)$ is the output vector of the system.



Lagrange Formula

- ✦ Let us consider a *Linear Time Invariant (LTI)* system in the form

$$\begin{aligned}\dot{x}(t) &= Ax(t) + Bu(t), & x(t_0) &= x_0 \\ y(t) &= Cx(t) + Du(t)\end{aligned}\tag{1}$$

The solution of the linear differential equation (1) defines the *time evolution of the state variables* and it is given by the Lagrange Formula

$$x(t) = e^{A(t-t_0)}x_0 + \int_{t_0}^t e^{A(t-\tau)}B u(\tau) d\tau, \quad t \geq t_0\tag{2}$$

- ✦ The *time evolution of the outputs* turns out to be

$$y(t) = Ce^{A(t-t_0)}x_0 + C \int_{t_0}^t e^{A(t-\tau)}B u(\tau) d\tau + D u(t), \quad t \geq t_0\tag{3}$$



Lagrange Formula

✧ Taking into account that

$$\frac{d}{dt} \int_{a(t)}^{b(t)} f(t, \tau) d\tau = f(t, b(t)) \frac{db(t)}{dt} - f(t, a(t)) \frac{da(t)}{dt} + \int_{a(t)}^{b(t)} \frac{d}{dt} f(t, \tau) d\tau$$

✧ Lagrange formula (2) can be easily verified by derivation (assuming $t_0 = 0$)

$$\begin{aligned} \dot{x}(t) &= \frac{d}{dt} (e^{At} x_0) + e^{A(t-t)} B u(t) + \int_0^t \frac{d}{dt} [e^{A(t-\tau)} B u(\tau)] d\tau \\ &= A e^{At} x_0 + B u(t) + \int_0^t A e^{A(t-\tau)} B u(\tau) d\tau \\ &= A \left[e^{At} x_0 + \int_0^t e^{A(t-\tau)} B u(\tau) d\tau \right] + B u(t) = A x(t) + B u(t) \end{aligned}$$



Free and forced evolution of LTI systems

- ✦ The *time evolution of the state and output variables* can be conceptually divided in two parts,

$$x(t) = e^{A(t-t_0)}x_0 + \int_{t_0}^t e^{A(t-\tau)}B u(\tau) d\tau, \quad t \geq t_0$$

Free evolution, $x_l(t)$ *Forced evolution, $x_f(t)$*

$$y(t) = C e^{A(t-t_0)}x_0 + C \int_{t_0}^t e^{A(t-\tau)}B u(\tau) d\tau + D u(t), \quad t \geq t_0$$

- ✦ The *free evolution* indicate the evolution of state and output vectors that would be obtained in the absence of input ($u(t) = 0$).
- ✦ The *forced evolution* indicate the evolution of state and output vectors that would be obtained in the presence of input and null initial conditions ($x_0 = 0$)



Free evolution: matrix A diagonalizable

✦ The free evolution of an LTI system in the time domain is defined by the matrix exponential e^{At} . Generalizing the Taylor expansion of an exponential to the matrix case, we have

$$e^M = \sum_{i=0}^{\infty} \frac{1}{i!} M^i = I_n + M + \frac{M^2}{2!} + \dots$$

✦ In case the matrix A has real and distinct eigenvalues, $\lambda_i, i=1, \dots, n$, it is diagonalizable: it is possible to find a matrix T_D , such that $\hat{A} = T_D A T_D^{-1} = \text{diag}\{\lambda_1, \dots, \lambda_n\}$. Then $A = T_D^{-1} \hat{A} T_D$ and e^{At} turns out to be

$$\begin{aligned} e^{At} &= \sum_{i=0}^{\infty} \frac{1}{i!} (A t)^i = T_D^{-1} \sum_{i=0}^{\infty} \frac{1}{i!} (\hat{A} t)^i T_D \\ &= T_D^{-1} \text{diag} \left\{ \sum_{i=0}^{\infty} \frac{(\lambda_1 t)^i}{i!}, \sum_{i=0}^{\infty} \frac{(\lambda_2 t)^i}{i!}, \dots, \sum_{i=0}^{\infty} \frac{(\lambda_n t)^i}{i!} \right\} \\ &= T_D^{-1} \text{diag}\{e^{\lambda_1 t}, e^{\lambda_2 t}, \dots, e^{\lambda_n t}\} T_D. \end{aligned}$$

Remark: $\lambda_1, \lambda_2 \dots \lambda_n$ are the eigenvalues of the A matrix, T_D is the transformation (i.e., $\hat{x} = T_D x$) that allows achieving a diagonal \hat{A} matrix.



Free evolution: aperiodic and pseudo-periodic modes

- ✦ The exponential terms,

$$e^{\lambda_i t}$$

are the modes of the system, named *aperiodic modes*

- ✦ In case of *complex conjugate eigenvalues* $\lambda_i = \alpha_i + j\omega_i$ e $\bar{\lambda}_i = \alpha_i - j\omega_i$, the corresponding complex exponential function determines a term as follows:

$$e^{\alpha_i t} \sin(\omega_i t + \varphi_i)$$

These latter modes are named *pseudo-periodic modes*.



Free evolution: A is not diagonalizable

- ✦ In case of a non-diagonalizable A matrix (eigenvalues with multiplicity (η) greater than one) we must resort to the Jordan form (see text for further information): the matrix \hat{A} has an almost diagonal structure, with the elements on the diagonal corresponding to the eigenvalues, with the addition of superdiagonal elements, Jordan miniblocks, which determine terms of the type

$$t^{\eta-1} e^{\lambda_i t}, \text{ if } \lambda_i \in \mathbb{R},$$

or

$$t^{\eta-1} e^{\alpha_i t} \sin(\omega_i t + \varphi_i) \text{ if } \lambda_i \in \mathbb{C}$$

where η is an integer between 1 and the maximum size of the Jordan miniblocks associated with λ_i .



Summary: free evolution of LTI system

- **Analysis in the time domain**

$$x_l(t) = e^{At} x_0$$

$$y_l(t) = C e^{At} x_0$$

- $x_1(t)$ (and then $y_1(t)$) is given by a combination of terms as

- $e^{\lambda_i t}$, in the case of real and distinct eigenvalues;

- $e^{\alpha_i t} \sin(\omega_i t + \varphi_i)$, in the case of complex conjugate eigenvalues of multiplicity one;

- $t^{\eta-1} e^{\lambda_i t}$, in the case of real eigenvalues with multiplicity $\eta > 1$;

- $t^{\eta-1} e^{\alpha_i t} \sin(\omega_i t + \varphi_i)$ in the case of complex conjugate eigenvalues of multiplicity $\eta > 1$.



Aperiodic evolution modes (1/3)

- ✦ *An aperiodic mode* is an evolution mode of a linear system related to a real eigenvalue of the matrix A of multiplicity 1. It can be written in the form

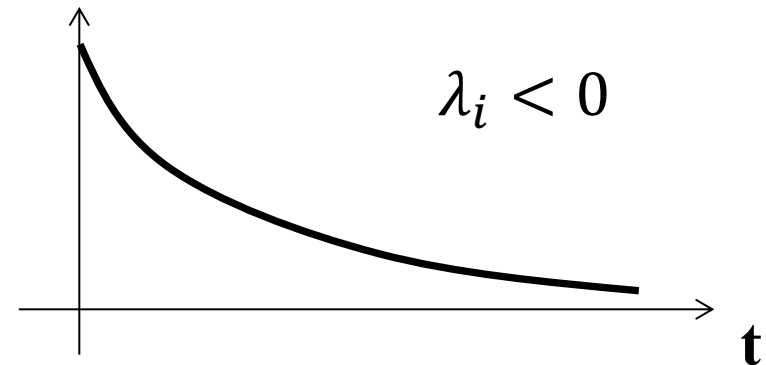
$$e^{\lambda_i t}$$

- ✦ Depending on the sign of the eigenvalue λ_i , an aperiod evolution mode can be
 - ✦ convergent ($\lambda_i < 0$)
 - ✦ constant ($\lambda_i = 0$)
 - ✦ divergent ($\lambda_i > 0$)

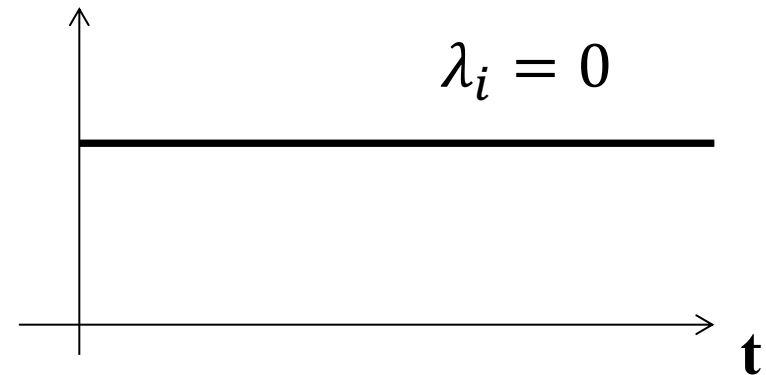


Aperiodic evolution modes (2/3)

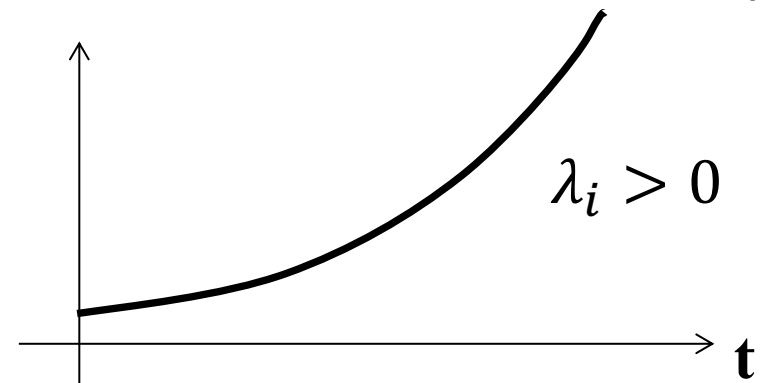
✦ *Convergent aperiodic mode*



✦ *Constant aperiodic mode*



✦ *Divergent aperiodic mode*





Aperiodic evolution modes (3/3)

- ✧ When the evolution mode is convergent it is possible to introduce a new parameter said *time constant of the mode* defined as

$$\tau_i = -\frac{1}{\lambda_i}$$

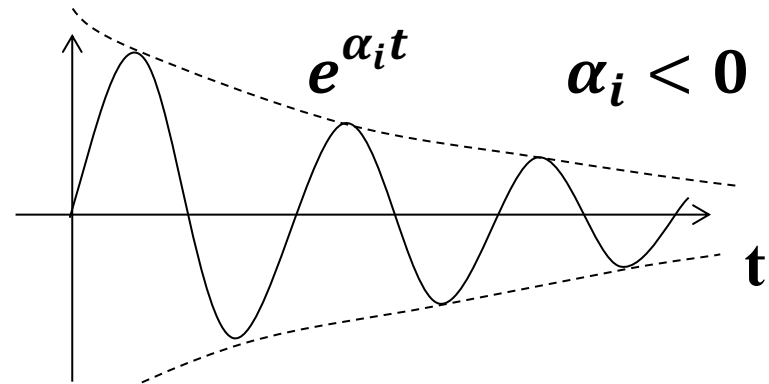
- ✧ The time constant gives us an information about the time needed before the convergent mode will be extinguished.
- ✧ It is straightforward to verify that
 - ✧ *After a time $\bar{t} = 3\tau$* the magnitude of the mode will be reduced to the 5% of the initial value
 - ✧ *After a time $\bar{t} = 4.6\tau$* the magnitude of the mode will be reduced to the 1% of the initial value



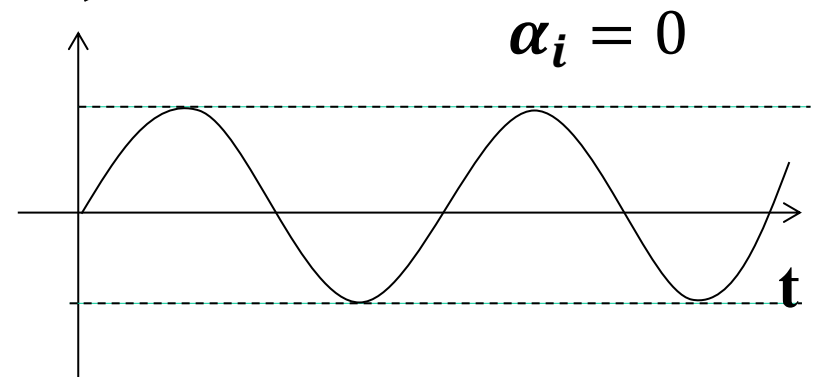
Pseudo-periodic evolution modes (1/5)

$$e^{\alpha_i t} \sin(\omega_i t + \varphi_i)$$

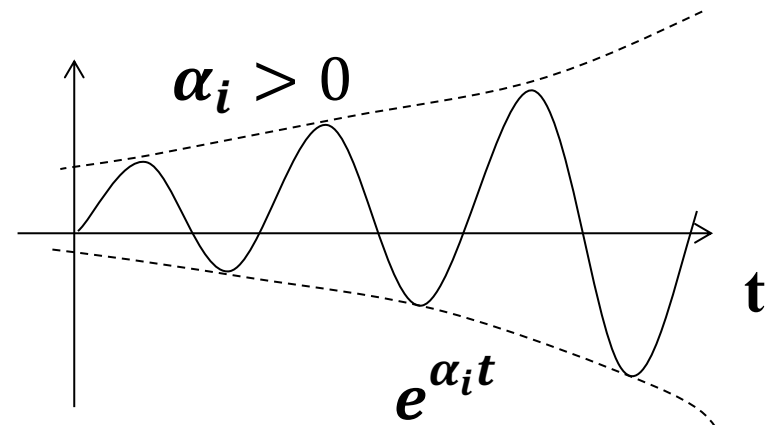
✦ *Convergent pseudo-periodic mode*



✦ *Constant pseudo-periodic mode*



✦ *Divergent pseudo-periodic mode*





Pseudo-periodic evolution modes (2/5)

- ✦ The pseudo-periodic mode is completely characterized by the pair of parameters (α_i, ω_i) that represent the real part and the imaginary part of the complex conjugate eigenvalues.
- ✦ These parameters give direct information both on the exponential law that envelops the oscillation peaks (parameter α_i) and on the angular frequency of the oscillations (parameter ω_i).
- ✦ Frequently, instead of using these parameters, other pairs of parameters related to (α_i, ω_i) through simple relations are used, and that provide information more directly related to other characteristics of the response, especially in the case of convergent oscillatory motion.



Pseudo-periodic evolution modes (3/5)

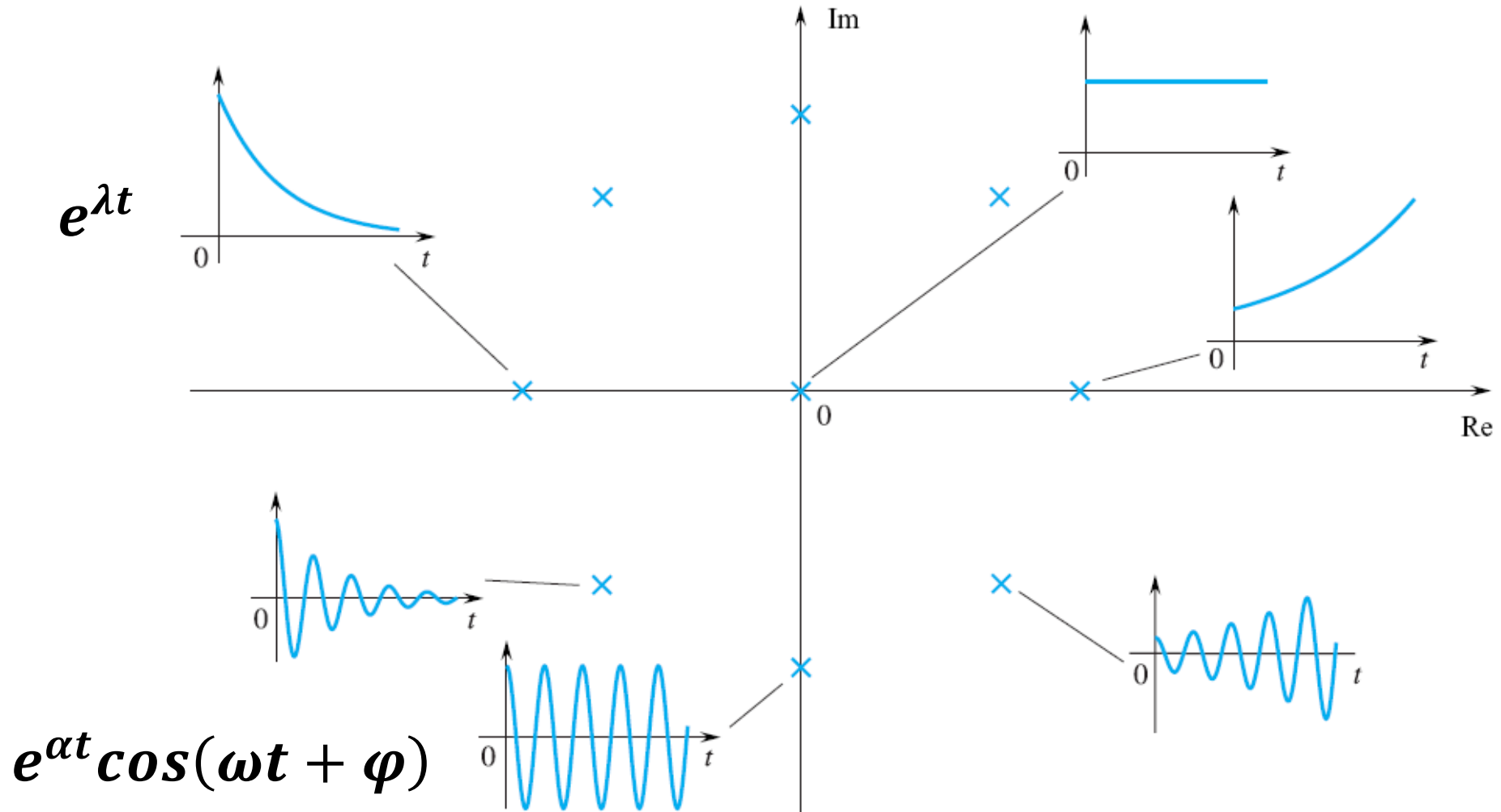
- ✧ For convergent pseudo-periodic mode, the *time constant* is defined as

$$\tau_i = -\frac{1}{\alpha_i}$$

- ✧ The parameter ω_i is called the angular frequency of the system, while T , related to ω_i by the relation $T=2\pi/\omega_i$, is called the oscillation period of the system.
- ✧ Sometimes, instead of the period T , the frequency $f=1/T$ is specified.



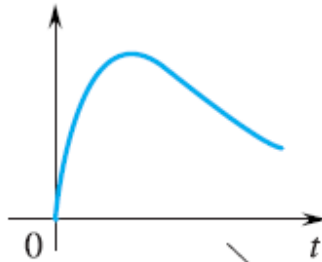
Evolution modes with distinct eigenvalues ($\eta=1$)





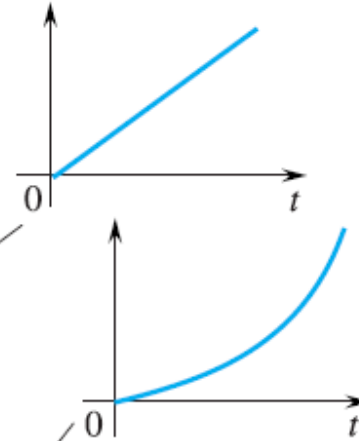
Evolution modes with multiple eigenvalues ($\eta=2$)

$$te^{\lambda t}$$

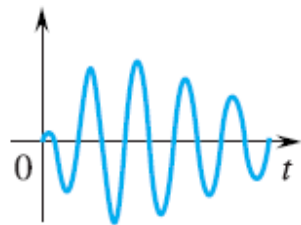


×

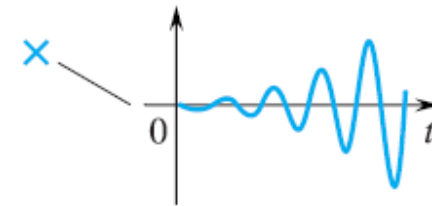
Im



Re



×



×

$$te^{\alpha t} \cos(\omega t + \varphi)$$

Natural frequency and damping factor

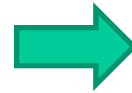
Other important parameters for pseudo-periodic mode are the **natural frequency** ω_n and the **damping coefficient** ζ .

The natural frequency is defined by

$$\omega_n^2 = \alpha^2 + \omega^2$$

and the damping factor

$$\zeta = -\frac{\alpha}{\sqrt{\alpha^2 + \omega^2}}$$



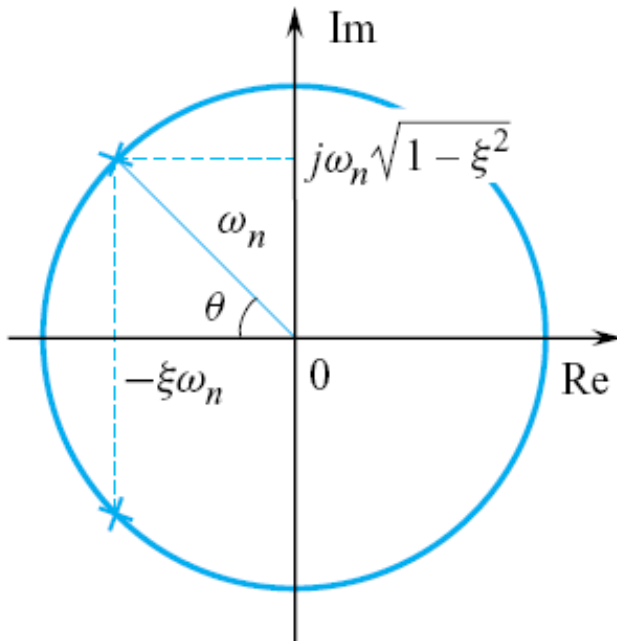
$$\alpha = -\zeta \omega_n,$$

$$\omega = \omega_n \sqrt{1 - \zeta^2}$$

$$\omega_n \cos \theta = \zeta \omega_n$$



$$\zeta = \cos \theta$$





Pseudo-periodic evolution modes (5/5)

- ✦ The *natural frequency* ω_n is the oscillation frequency of the pseudo-periodic mode when $\alpha = 0$.
- ✦ For *convergent* pseudo-periodic modes the *damping coefficient* $\zeta \in (0,1]$ while *for divergent pseudo-periodic modes* $\zeta \in [-1,0)$
- ✦ For *convergent* pseudo-periodic modes, the *damping coefficient* ζ relates the oscillations of the pseudo-periodic mode to the time before the evolution will extinguish. For $\zeta \ll 1$

$$\zeta = -\frac{\alpha}{\omega_n} \cong -\frac{\alpha}{\omega} = \frac{T}{2\pi\tau} \ll 1$$

where T is the oscillation period. Indeed, the number of the oscillation before the mode will extinguish increases when ζ becomes small.

$$\zeta = \frac{T}{2\pi\tau} \cong \frac{T}{6\tau} \quad \longrightarrow \quad \frac{1}{2\zeta} \cong \frac{3\tau}{T} \quad \# \text{ of oscillations before the mode will extinguish}$$



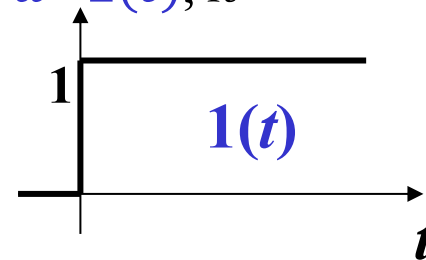
Forced response in the time domain

- Let us consider the forced response of an LTI system in the output ($x_0 = 0$)

$$y_f(t) = C \int_0^t e^{A(t-\tau)} B u(\tau) d\tau + D u(t), \quad t \geq t_0$$

- The evaluation of the forced response in the time domain is demanding due to the presence of the convolution product.
- Only in some particular case, such as the *step response* $u(t) = \bar{u} \cdot \mathbf{1}(t)$, it becomes straightforward

$$\begin{aligned} y_f(t) &= C \int_0^t e^{A(t-\tau)} B \bar{u} d\tau + D \bar{u} \\ &= \left[-CA^{-1} e^{A(t-\tau)} B \bar{u} \right]_0^t + D \bar{u} \\ &= CA^{-1} e^{At} B \bar{u} + [-CA^{-1} B + D] \bar{u} \end{aligned}$$



- In the other cases the forced response is evaluated in the *Laplace domain*



First-order LTI system

✦ For a **first-order LTI system** in the form

$$\begin{aligned} \dot{x}(t) &= ax(t) + bu(t) \\ y(t) &= cx(t) + du(t), \end{aligned} \quad x(t_0 = 0) = x_0$$

the *time evolution of the state variable* in case of $\mathbf{u}(t) = \bar{\mathbf{u}} \cdot \mathbf{1}(t)$ is given

$$x(t) = \underbrace{e^{at} x_0}_{\mathbf{x}_l} + \underbrace{\frac{1}{a} e^{at} b \bar{u} - \frac{1}{a} b \bar{u}}_{\mathbf{x}_f}, \quad t \geq 0$$

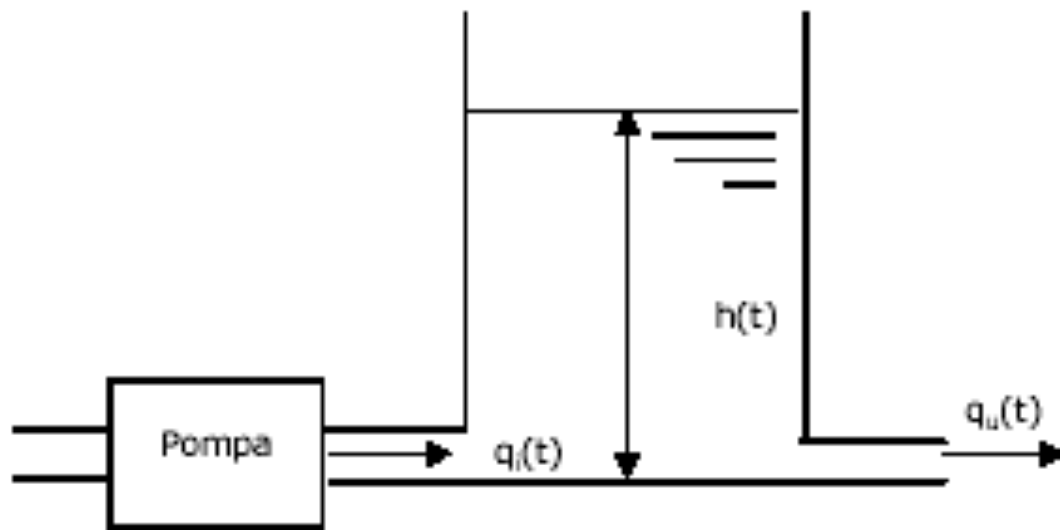
✦ The *time evolution of the output* turns out to be

$$y(t) = ce^{at} x_0 + c \frac{1}{a} e^{at} b \bar{u} - c \frac{1}{a} b \bar{u} + d \bar{u},$$

$$= \underbrace{ce^{at} x_0}_{\mathbf{y}_l} - \underbrace{c \frac{b}{a} \bar{u} (1 - e^{at})}_{\mathbf{y}_f} + d \bar{u}$$



Example of first order LTI system:



input: $u(t) = q_i(t)$

output: $y(t) = h(t)$

state: $x(t) = h(t)$

SS representation:

$$\dot{x}(t) = -\frac{k}{S}x(t) + \frac{1}{S}u(t)$$

$$y(t) = x(t)$$

⤴ For a first order LTI, the *time evolution of the output for $u(t) = \bar{u} \cdot \mathbf{1}(t)$ is given by*

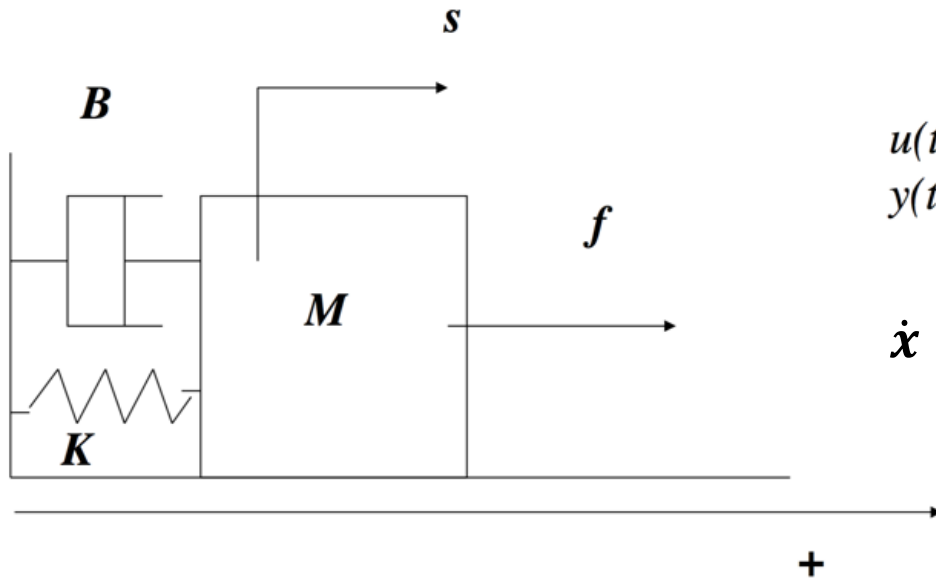
$$y(t) = \underbrace{ce^{at}x_0}_{y_l} - \underbrace{c\frac{b}{a}\bar{u}(1 - e^{at}) + d\bar{u}}_{y_f}$$

⤴ In this case, $a = -\frac{k}{S}$, $b = \frac{1}{S}$, $c = 1$, $d = 0$,

$$y_l(t) = e^{-\frac{k}{S}t}x_0 = e^{-\frac{t}{\tau}}x_0, \text{ with } \tau = -\frac{1}{a} = \left(\frac{S}{k}\right) \quad y_f(t) = \left(\frac{1}{k}\right)\bar{u}(1 - e^{-\frac{k}{S}t})$$

time constant
static gain

Example of second-order LTI system: mass-spring-damper system



$$u(t) = f(t)$$

$$y(t) = s(t)$$

- State space representation

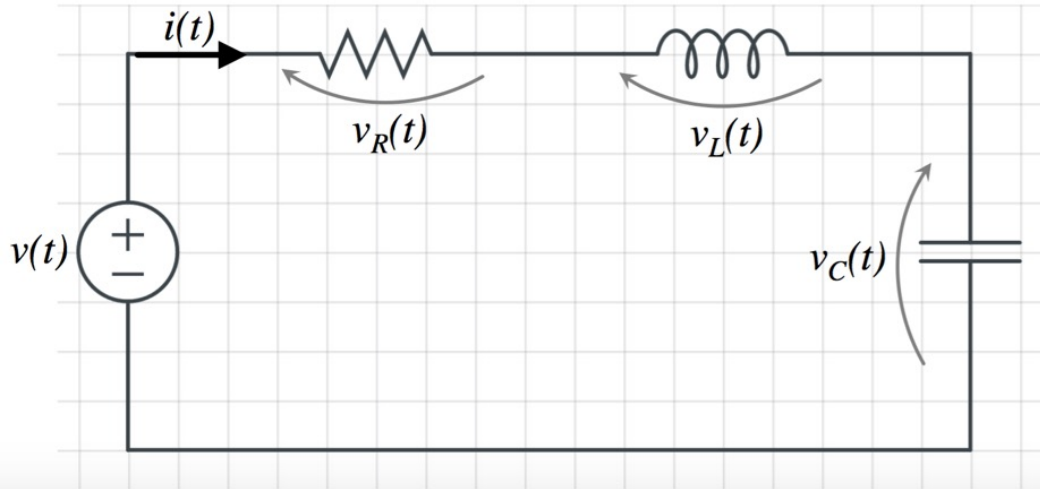
$$\mathbf{x}_1 = s, \mathbf{x}_2 = \frac{ds}{dt} = \dot{s} = \dot{\mathbf{x}}_1$$

$$\dot{\mathbf{x}} = \begin{pmatrix} \dot{\mathbf{x}}_1 \\ \dot{\mathbf{x}}_2 \end{pmatrix} = \begin{pmatrix} 0 & 1 \\ -\frac{K}{M} & -\frac{B}{M} \end{pmatrix} \begin{pmatrix} \mathbf{x}_1 \\ \mathbf{x}_2 \end{pmatrix} + \begin{pmatrix} 0 \\ 1/M \end{pmatrix} u,$$

$$\mathbf{y} = \begin{pmatrix} 1 & 0 \end{pmatrix} \begin{pmatrix} \mathbf{x}_1 \\ \mathbf{x}_2 \end{pmatrix}$$

Problem:

For the mass-spring-damper system reported in figure, assuming $M=1\text{Kg}$, $K=16\text{ N/m}$, evaluate the evolution modes by varying B values $[\text{Ns/m}]$ in the interval $[0\ 20]$.



$$\mathbf{u}(t) = \mathbf{v}(t), \quad \mathbf{y}(t) = \mathbf{v}_c(t)$$

$$\mathbf{x}_1(t) = \mathbf{v}_c(t) \quad \mathbf{x}_2(t) = \mathbf{i}_L(t)$$

Problem:

For the RLC circuit in series configuration, **compute the input-output and state space representations.**

Assuming the capacity value $C=1e-6 \mu\text{F}$, and the inductance value $L=1e-3 \text{ mH}$, compute the values of \mathbf{R} for which the system exhibits aperiodic and pseudo-periodic modes.



Appendix 1

INVERSE OF A MATRIX $N \times N$



Inverse of a matrix

✧ Given a quadratic and invertible matrix

$$A = \begin{pmatrix} x_{1,1} & \dots & x_{1,j} \\ \vdots & \ddots & \vdots \\ x_{i,1} & \dots & x_{i,j} \end{pmatrix}$$

its inverse is defined as

$$A^{-1} = \frac{1}{\det(A)} \begin{pmatrix} \text{cof}(A, x_{1,1}) & \dots & \text{cof}(A, x_{1,j}) \\ \vdots & \ddots & \vdots \\ \text{cof}(A, x_{i,1}) & \dots & \text{cof}(A, x_{i,j}) \end{pmatrix}^T$$

where the cofactor is

$$\text{cof}(A, i, j) = (-1)^{i+j} \det(\text{minor}(A, i, j))$$

and the minor (i, j) is the determinant of the matrix obtained excluding the row i and the column j .



Inverse of a 2x2 matrix

✦ Given a matrix

$$A = \begin{pmatrix} a & b \\ c & d \end{pmatrix}$$

its inverse is

$$A^{-1} = \frac{1}{ad - bc} \begin{pmatrix} d & -b \\ -c & a \end{pmatrix}$$



Inverse of a 3×3 matrix

✧ Given a matrix

$$A = \begin{pmatrix} A_{11} & A_{12} & A_{13} \\ A_{21} & A_{22} & A_{23} \\ A_{31} & A_{32} & A_{33} \end{pmatrix}$$

its inverse is

$$\frac{1}{\det(A)} \begin{pmatrix} + \begin{vmatrix} A_{22} & A_{23} \\ A_{32} & A_{33} \end{vmatrix} & - \begin{vmatrix} A_{12} & A_{13} \\ A_{32} & A_{33} \end{vmatrix} & + \begin{vmatrix} A_{12} & A_{13} \\ A_{22} & A_{23} \end{vmatrix} \\ - \begin{vmatrix} A_{21} & A_{23} \\ A_{31} & A_{33} \end{vmatrix} & + \begin{vmatrix} A_{11} & A_{13} \\ A_{31} & A_{33} \end{vmatrix} & - \begin{vmatrix} A_{11} & A_{13} \\ A_{21} & A_{23} \end{vmatrix} \\ + \begin{vmatrix} A_{21} & A_{22} \\ A_{31} & A_{32} \end{vmatrix} & - \begin{vmatrix} A_{11} & A_{12} \\ A_{31} & A_{32} \end{vmatrix} & + \begin{vmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{vmatrix} \end{pmatrix}$$



Appendix 2

EIGENVALUES AND EIGENVECTORS



Eigenvalues and eigenvectors

- Given a matrix $A \in R^{n \times n}$, a scalar $\lambda \in C$ is said *eigenvalue* of the matrix A if there exists a vector $v \in C^n$, said *eigenvector*, such that

$$Av = \lambda v$$

- Taking into account account that eigenvalues and eigenvectors of a matrix verify the equation

$$(A - \lambda I)v = 0.$$

The eigenvalues can be found evaluating the roots of the *characteristic polynomial* $p(\lambda)$ defined as

$$p(\lambda) = \det(A - \lambda I).$$



Examples

$$A = \begin{pmatrix} 1 & 2 \\ 3 & -4 \end{pmatrix}$$

$$p(\lambda) = \lambda^2 + 3\lambda - 10$$

Eigenvalues

$$\lambda_1 = 2, \lambda_2 = -5$$

$$A = \begin{pmatrix} 0 & -1 \\ 1 & -2 \end{pmatrix}$$

$$p(\lambda) = \lambda^2 + 2\lambda + 1$$

Eigenvalues

$$\lambda_1 = \lambda_2 = -1$$